



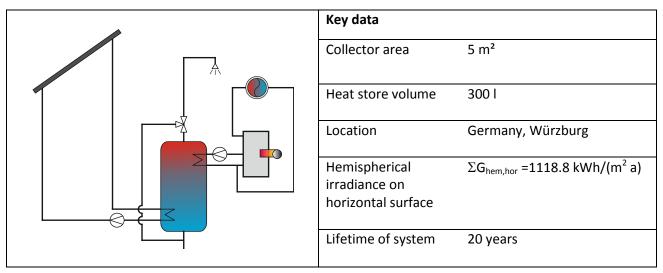
Reference System, Germany Solar Domestic Hot Water System for Single-Family House

Description:	Definition of the reference solar domestic hot water (SDHW) system, Germany
Date:	23.03.2018, revised 10.04.2018 ¹
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Download possible at:	http://task54.iea-shc.org/

Introduction

This document describes the reference solar domestic hot water (SDHW) system for domestic hot water preparation and space heating in Germany. The system is modelled with TRNSYS to calculate the fuel consumption and electric energy needed to provide the required domestic hot water and space heating as well as the substituted fuel provided by the SDHW system. Using this result the levelized costs of heating (LCOH) for the substituted fuel is calculated using eq. 1 and the reference costs for the investment of the system, installation costs, fuel and electricity costs.

Hydraulic Scheme of the System



Levelized Cost of Heat (LCoH)

LCoHs solar part without VAT	0.141 €/kWh
LCoHc conventional part without VAT	0.116 €/kWh
LCoHo complete system without VAT	0.122 €/kWh





Reference System, Germany Solar Domestic Hot Water System for Single-Family House

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Details of the system

Type of systemSolar domestic hot water (SDHW) systemWeather data includingtest reference year (TRY Wirzburg)- Hemispherical irradiance on horizontal surface $\Sigma G_{hem,hor} = 1118.8 kWh/(m^2 a)$ - Beam irradiance on horizontal surface $\Sigma G_{hem,hor} = 568.7 kWh/m^2 a)$ - Diffuse irradiance on horizontal surface $\Sigma G_{diff,hor} = 568.7 kWh/m^2 a)$ - Ambient temperature $T_{omb,av} = 9.0 ^{\circ}C$ in hourly values $South = 0^{\circ}$ - Collector orientation $45 ^{\circ}$ - South deviation of collectorsouth = 0°- Ground reflectance 0.2 - Resulting hemispherical irradiance on tiltedsurface $\Sigma G_{hem,ulit} = 1229.8 kWh/(m^2 a)$ Load information including- Heat demand space heating9090 kWh/a /1/- Tapping profileEU-tapping profile L (4254 kWh/a) /2/- Tapping temperature 55° caccording EU tapping profile- Average inlet temperature of cold water $10^{\circ}C$ - Collector information based on gross areaTRNSYS-type 132Number of collectors2Collector area of one collector $2.5 m^2$ Maximum collector efficiency 0.684 Incidence angle modifier for diffuse irradiance b_0 0.2 Incidence angle modifier for diffuse irradiance b_0 <td< th=""><th>Location</th><th>Germany, Würzburg</th></td<>	Location	Germany, Würzburg
Weather data includingtest reference year (TRY Würzburg)- Hemispherical irradiance on horizontal surface $\Sigma G_{hem,hor} = 1118.8 kWh/(m^2 a)$ - Beam irradiance on horizontal surface $\Sigma G_{hem,hor} = 550.1 kWh/(m^2 a)$ - Ambient temperature $\Sigma G_{diff,hor} = 558.7 kWh/m^2 a)$ - Ambient temperature $T_{amb,av} = 9.0 \ ^{\circ}C$ - Ambient temperature $45 \ ^{\circ}$ - Collector vilt angle to horizontal $45 \ ^{\circ}$ - South deviation of collectorsouth = 0°- Ground reflectance 0.2 - Resulting hemispherical irradiance on tilted $\Sigma G_{hem,tilt} = 1229.8 kWh/(m^2 a)$ Load information including EU -tapping profile L (4254 kWh/a) /2/- Tapping temperature $55^{\circ}C$ according EU tapping profile- Average inlet temperature of cold water $10^{\circ}C$ - Collector s 2 Collector efficiency 0.684 Number of collectors 2 Collector area of one collector $2.5 \ m^2$ Maximum collector efficiency 0.684 Incidence angle modifier for diffuse irradiance k_d 0.91 Linear heat loss coefficient a_2 $0.011 \ W/(m^2 k^2)$ Effective heat capacity C_{eff} $8.0 \ k/(m^2 k)$ Heat store parameters $TRNSYS-type 340$ Heat store volume $300 \ 1$ Auxiliary volume for DHW preparation $150 \ 1$ Store inner diameter $0.6 \ m$ Rel. height of solar inlet 0.4 Rel. height of solar inlet 0.4	Type of system	
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- Resulting hemispherical irradiance on tilted surface $\Sigma G_{hem,tilt} = 1229.8 \text{ kWh/(m^2 a)}$ Load information including $\Sigma G_{hem,tilt} = 1229.8 \text{ kWh/(m^2 a)}$ - Heat demand space heating9090 kWh/a /1/- Tapping profileEU-tapping profile L (4254 kWh/a) /2/- Tapping temperature55°C according EU tapping profile- Average inlet temperature of cold water10°C- Cold water inlet temperature amplitude0 KCollector information based on gross areaTRNSYS-type 132Number of collectors2Collector area of one collector2.5 m²Maximum collector efficiency0.684Incidence angle modifier for direct irradiance b_00.2Incidence angle modifier for diffuse irradiance K_d0.91Linear heat loss coefficient a_13.51 W/(m²K)2nd order heat loss coefficient a_20.011 W/(m²K2)Effective heat capacity C _{eff} 8.0 kJ/(m²K)Heat store parametersTRNSYS-type 340Heat store volume300 IAuxiliary volume for DHW preparation150 IStore inner diameter0.6 mRel. height of solar outlet0.04Rel. height of solar outlet0.04Rel. height of solar outlet0.8	- South deviation of collector	south = 0°
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- Tapping temperature55°C according EU tapping profile- Average inlet temperature of cold water10°C- Cold water inlet temperature amplitude0 KCollector information based on gross areaTRNSYS-type 132Number of collectors2Collector area of one collector2.5 m²Maximum collector efficiency0.684Incidence angle modifier for direct irradiance b00.2Incidence angle modifier for diffuse irradiance Kd0.91Linear heat loss coefficient a13.51 W/(m²K)2nd order heat loss coefficient a20.011 W/(m²K²)Effective heat capacity ceff8.0 kJ/(m²K)Heat store parametersTRNSYS-type 340Heat store volume300 lAuxiliary volume for DHW preparation150 lStore inner diameter0.6 mRel. height of solar unlet0.04Rel. height of solar outlet0.04Rel. height of auxiliary inlet0.8	- Heat demand space heating	9090 kWh/a /1/
- Average inlet temperature of cold water10°C- Cold water inlet temperature amplitude0 KCollector information based on gross areaTRNSYS-type 132Number of collectors2Collector area of one collector2.5 m²Maximum collector efficiency0.684Incidence angle modifier for direct irradiance b00.2Incidence angle modifier for diffuse irradiance Kd0.91Linear heat loss coefficient a13.51 W/(m²K)2nd order heat loss coefficient a20.011 W/(m²K²)Effective heat capacity Ceff8.0 kJ/(m²K)Heat store parametersTRNSYS-type 340Heat store volume300 lAuxiliary volume for DHW preparation150 lStore inner diameter0.6 mRel. height of solar outlet0.04Rel. height of solar outlet0.04Rel. height of auxiliary inlet0.8	- Tapping profile	EU-tapping profile L (4254 kWh/a) /2/
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Collector information based on gross areaTRNSYS-type 132Number of collectors2Collector area of one collector2.5 m²Maximum collector efficiency0.684Incidence angle modifier for direct irradiance b₀0.2Incidence angle modifier for diffuse irradiance Kd0.91Linear heat loss coefficient a₁3.51 W/(m²K)2nd order heat loss coefficient a₂0.011 W/(m²K²)Effective heat capacity ceff8.0 kJ/(m²K)Heat store parametersTRNSYS-type 340Heat store volume300 lAuxiliary volume for DHW preparation150 lStore inner diameter0.6 mRel. height of solar outlet0.04Rel. height of auxiliary inlet0.8	- Average inlet temperature of cold water	10°C
Number of collectors2Collector area of one collector2.5 m²Maximum collector efficiency0.684Incidence angle modifier for direct irradiance b₀0.2Incidence angle modifier for diffuse irradiance Kd0.91Linear heat loss coefficient a₁3.51 W/(m²K)2nd order heat loss coefficient a₂0.011 W/(m²K²)Effective heat capacity ceff8.0 kJ/(m²K)Heat store parametersTRNSYS-type 340Heat store volume300 lAuxiliary volume for DHW preparation150 lStore inner diameter0.6 mRel. height of solar outlet0.04Rel. height of auxiliary inlet0.8	- Cold water inlet temperature amplitude	0 K
Collector area of one collector2.5 m²Maximum collector efficiency0.684Incidence angle modifier for direct irradiance b₀0.2Incidence angle modifier for diffuse irradiance Kd0.91Linear heat loss coefficient a₁3.51 W/(m²K)2nd order heat loss coefficient a₂0.011 W/(m²K²)Effective heat capacity ceff8.0 kJ/(m²K)Heat store parametersTRNSYS-type 340Heat store volume300 lAuxiliary volume for DHW preparation150 lStore inner diameter0.6 mRel. height of solar outlet0.04Rel. height of auxiliary inlet0.8		TRNSYS-type 132
Maximum collector efficiency0.684Incidence angle modifier for direct irradiance b00.2Incidence angle modifier for diffuse irradiance Kd0.91Linear heat loss coefficient a13.51 W/(m²K)2nd order heat loss coefficient a20.011 W/(m²K²)Effective heat capacity ceff8.0 kJ/(m²K)Heat store parametersTRNSYS-type 340Heat store volume300 lAuxiliary volume for DHW preparation150 lStore inner diameter0.6 mRel. height of solar outlet0.04Rel. height of auxiliary inlet0.8		
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Incidence angle modifier for diffuse irradiance Kd0.91Linear heat loss coefficient a13.51 W/(m²K)2nd order heat loss coefficient a20.011 W/(m²K²)Effective heat capacity ceff8.0 kJ/(m²K)Heat store parametersTRNSYS-type 340Heat store volume300 lAuxiliary volume for DHW preparation150 lStore inner diameter0.6 mRel. height of solar inlet0.04Rel. height of solar outlet0.8	Maximum collector efficiency	0.684
Linear heat loss coefficient a13.51 W/(m²K)2nd order heat loss coefficient a20.011 W/(m²K²)Effective heat capacity ceff8.0 kJ/(m²K)Heat store parametersTRNSYS-type 340Heat store volume300 lAuxiliary volume for DHW preparation150 lStore inner diameter0.6 mRel. height of solar inlet0.4Rel. height of solar outlet0.04Rel. height of auxiliary inlet0.8	Incidence angle modifier for direct irradiance b ₀	0.2
2nd order heat loss coefficient a20.011 W/(m²K²)Effective heat capacity ceff8.0 kJ/(m²K)Heat store parametersTRNSYS-type 340Heat store volume300 lAuxiliary volume for DHW preparation150 lStore inner diameter0.6 mRel. height of solar inlet0.04Rel. height of auxiliary inlet0.8		0.91
Effective heat capacity c _{eff} 8.0 kJ/(m²K)Heat store parametersTRNSYS-type 340Heat store volume300 lAuxiliary volume for DHW preparation150 lStore inner diameter0.6 mRel. height of solar inlet0.4Rel. height of solar outlet0.04Rel. height of auxiliary inlet0.8	Linear heat loss coefficient a ₁	3.51 W/(m²K)
Heat store parametersTRNSYS-type 340Heat store volume300 lAuxiliary volume for DHW preparation150 lStore inner diameter0.6 mRel. height of solar inlet0.4Rel. height of solar outlet0.04Rel. height of auxiliary inlet0.8	2nd order heat loss coefficient a ₂	
Heat store volume300 IAuxiliary volume for DHW preparation150 IStore inner diameter0.6 mRel. height of solar inlet0.4Rel. height of solar outlet0.04Rel. height of auxiliary inlet0.8	Effective heat capacity c _{eff}	
Auxiliary volume for DHW preparation150 IStore inner diameter0.6 mRel. height of solar inlet0.4Rel. height of solar outlet0.04Rel. height of auxiliary inlet0.8	Heat store parameters	TRNSYS-type 340
Store inner diameter0.6 mRel. height of solar inlet0.4Rel. height of solar outlet0.04Rel. height of auxiliary inlet0.8	Heat store volume	300 l
Rel. height of solar inlet0.4Rel. height of solar outlet0.04Rel. height of auxiliary inlet0.8	Auxiliary volume for DHW preparation	
Rel. height of solar outlet0.04Rel. height of auxiliary inlet0.8	Store inner diameter	0.6 m
Rel. height of auxiliary inlet0.8	Rel. height of solar inlet	0.4
	Rel. height of solar outlet	0.04
Rel height of auxiliary outlet	Rel. height of auxiliary inlet	0.8
	Rel. height of auxiliary outlet	0.5
Rel. height of sensor for collector loop0.2	Rel. height of sensor for collector loop	0.2
Rel. height of sensor for auxiliary heating0.7	Rel. height of sensor for auxiliary heating	0.7
Set temperature for DHW 57.5 °C +- 2.5 K	Set temperature for DHW	57.5 °C +- 2.5 K
Overall heat loss capacity rate of store 3.56 W/K	Overall heat loss capacity rate of store	3.56 W/K
Effective vertical conductivity 1.2 W/(mK)	Effective vertical conductivity	1.2 W/(mK)





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Heat transfer capacity rate of solar loop HX	$(kA)_{WT,Sol} = 102,7 \cdot \dot{m}^{0,226} \cdot \vartheta_m^{0,550} [W/K]$
Heat transfer capacity rate of auxiliary loop HX	$(kA)_{WT,Aux} = 82,3 \cdot \dot{m}^{0,185} \cdot \vartheta_m^{0,482} [W/K]$
Volume solar loop HX	101
Volume auxiliary loop HX	61
Maximum heat store temperature	90 °C
Ambient temperature of heat store	15 °C
Solar thermal controller and hydraulic piping	
Total pipe length of collector loop	20 m
Inner diameter of collector loop pipe	16 mm
Ambient temperature of heat store	15 °C
Mass flow collector loop	40 kg/(m ² h), constant
Temperature difference collector start-up	6 K
Temperature difference collector shut-off	4 К
Electric power of solar thermal controller	3 W
Operating hours of solar thermal controller per year	8760 h
Electric consumption of controller per year	26.3 kWh
Electric power of solar loop pump	40 W
Operating hours of solar loop pump	1277 h (L-profile), 929 h (M-profile)
Electric consumption of solar loop pump per year	51.1 kWh (L-profile), 37.2 kWh (M-profile)
Conventional back up system	
Type of auxiliary heating	Gas condensing boiler
Boiler capacity	19 kW
Mass flow	1090 kg/h (∆T = 15 K)
Efficiency factor of boiler	0.9
Electric power of controller	3 W
Operating hours of controller per year	8760
Electric consumption of controller per year	26.3 kWh
Electric power of pump	55 W
Operating hours of pump (aux. heating + space	3999 h
heating)	
Electric consumption of pump per year	220 kWh
Investment costs solar thermal system	
Solar thermal collector, heat store, solar thermal	3600 € /5/
controller solar thermal hydraulic components	
Installation	1250 € /5/
Credit conventional heat store and share of	-1000€
installation	
Overall investment costs solar thermal part I ₀	3850€
Operation costs conventional part per year	
Heat demand hot water	3002 kWh/a
Fuel demand hot water	3335 kWh/a
Heat demand space heating	9090 kWh/a /1/
Fuel demand space heating	10100 kWh/a





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Fuel demand hot water + space heating E _t	13435 kWh/a
Cost per kWh fuel (gas)	0.066 € kWh/a /4/
Fuel costs	887 €/a
Electricity demand	246 kWh/a
Cost per kWh electric energy /4/	0.254 €
Electricity costs	63 €/a
Maintenance costs	200 €/a /3/
Gas meter	130 €/a /3/
Yearly operation and maintenance cost conventional	1279€
part C _t	
Operation costs solar part per year	
Electricity demand	83 kWh/a
Cost per kWh electric energy /4/	0.254 €
Electricity costs	21 €/a
Maintenance costs (I ₀ * 2%)	100 €/a
Yearly operation and maintenance cost solar part Ct	121 €/a
Fractional energy savings with credit for 150l-store,	40 %
UA=2,05 W/K	
Saved final energy (year t) E _t	2226 kWh
Type of incentives	None
Amount of incentives	0€
Lifetime of system	20 year
Discount rate r	0 %
Inflation rate	0 %
Corporate tax rate TR	0 %
Asset depreciation (year t) DEP _t	0€
Subsidies and incentives (year t) S_t (considered in I_0)	0€
Residual value RV	0€
Discount rate r	0%
VAT rate	19 %





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Calculation of levelized cost LCoH:

$$LCoH = \frac{I_0 - S_0 + \sum_{t=1}^{T} \frac{C_t (1 - TR) - DEP_t \cdot TR}{(1 + r)^t} - \frac{RV}{(1 + r)^T}}{\sum_{t=1}^{T} \frac{E_t}{(1 + r)^t}}$$

Where:

LCoH: levelized cost of heat in €/kWh I_0 : initial investment in € S₀: subsidies and incentives in € C_t : operation and maintenance costs (year t) in € *TR*: corporate tax rate in % DEP_t : asset depreciation (year t) in \in RV: residual value in \in E_t : saved final energy (year t) in kWh r: discount rate in % T: period of analysis in years

Annex: Comparison to figures published in Solar Heat Worldwide

To compare the above presented LCoHs based on the saved final energy with the $LCoH_{SHWW}$ presented in Solar Heat World Wide based on the collector yield the following table is presented

Collector yield (year t) E _t	2288 kWh
LCoH _{shww} solar part without VAT	0.105 €/kWh

References

[1] EN 12977-2 (2012): "Thermal solar systems and components – Custom built systems – Part 2: Test methods for solar water heaters and combisystems".

- [2] COMMISSION DELEGATED REGULATION (EU) No 812/2013, ANNEX VII.
- [3] Hafner, B. (2016): "E-Mail". Dated 13.06.2016.
- [4] Check24 (2016): "Würzburg reference costs". URL: <u>www.check24.com</u> (accessed in Sept. 2016).
- [5] Mean values of evaluated invoices, supplied by Bafa.

[6] Louvet, Y., Fischer, S. et. al. (2017): *"IEA SHC Task 54 Info Sheet A1: Guideline for levelized cost of heat (LCOH) calculations for solar thermal applications"*. URL: <u>http://task54.iea-shc.org/.</u>

[7] Louvet, Y., Fischer, S. et.al. (2017): "Entwicklung einer Richtlinie für die Wirtschaftlichkeitsberechnung solarthermischer Anlagen: die LCoH Methode". Symposium Thermische Solarenergie, Bad Staffelstein.





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¹ To avoid confusion with the results of other works ([1], [8], [9]) also using the notion of LCoH for solar thermal systems, new acronyms were introduced in this Info Sheet. As previous studies have considered different assumptions for the definition of the terms of the LCoH equation, it does not make sense to compare the values they obtained with the LCoHs, LCoHc and LCoHo values defined here. A detailed explanation of the differences between the approaches chosen in the framework of IEA-SHC Task 54 and in the Solar Heat Worldwide report [9] can be found in Info Sheet A13 [10].